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⑤④ Lever arrangement for automatic pistol for positioning firing pin and for decocking.

⑤⑦ A device or arrangement for automatic firearms in which a transverse rotatable element (10) is mounted in the slide (5) and is operable to move by camming action the firing pin (16) partially forward to avoid being struck by the hammer (6), all prior to its further functioning to rotate the sear (17) to release the hammer (6). The device also prevents further movement forward of the firing pin (16) during rotation and the device may include means to hold it in its firing-pin-forward position.

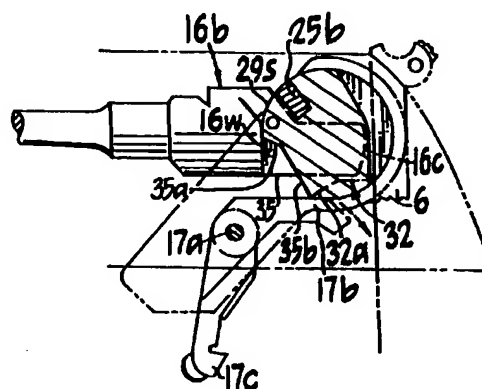


FIG. 3b

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Background of the Invention

Prior automatic firearms have included arrangements for moving the firing pin forward in an effort to prevent the hammer engaging the pin as it descends (U.S. Patent No. 4,589,327). It has also been proposed to rotate levers to move a protrusion to a position to protect the firing pin from engaging the hammer as the hammer moves toward its down position (U. S. Patent No. 4,590,697).

However, none of these arrangements has provided or suggested a compact decocking lever arrangement which is capable, as initially rotated, of moving the firing pin forward a limited distance, preventing further movement forward and upon further rotation of the lever, causing the sear to release the hammer.

Summary of the Invention

Broadly, the present invention is a device or arrangement rotatable transversely through the slide including a hand rotatable lever which, as initially rotated, causes an internal first cam surface to urge the firing pin forward and also has a second cam surface which upon further rotation engages the sear to rotate. Sear rotation releases the hammer from its cocked position allowing the hammer to fall into contact with the stop surface at the bottom of the hammer slot in the slide.

It is also a feature of the arrangement that it may be spring urged to return to its rest position upon release by the operator.

It is also a feature that the device not only moves the firing pin forward but also prevents any additional firing pin movement forward by action of the hammer as the hammer is lowered.

Brief Description of the Drawings

Fig. 1a is a side elevational view of the left side of the pistol including the left portion of the device;

Fig. 1b is an enlarged partial plan view of the rear portion of the pistol;

Fig. 1c is a rear end view of the slide with the device in its nonoperating position;

Fig. 2 is an exploded perspective view of the two interconnectable device portions and certain associated parts;

Fig. 3a is a partial elevational view of the right lever portion, sear, and slide;

Fig. 3b is a view similar to Fig. 3a with the right portion in the full activated position;

Fig. 4a is a partial elevational view of left lever portion and firing pin with the device shown both in its inactive and its active positions; and

Fig. 5 is a side elevational view showing a spring

being loaded as the device is held in the active turned-down position.

Description of the Preferred Embodiment

In Figs. 1a-1c, pistol 1 includes barrel 2, frame 3, grip 4, slide 5, hammer 6, trigger guard 7, trigger 8 and rear sight 9. Also shown is device 10 positioned in round slide side holes 5a, 5b. Fig. 1b shows rear slide cutout slot 14 having surfaces 14a, 14b parallel to barrel 2 and hammer lower seating surface 15b. Also shown is surface 15a. Lower surface 15b is rearward of upper surface 15a a few thousandths of an inch though this small dimensional difference is not shown in Fig. 1b. Protruding rear firing pin portion 16c of firing pin 16 is shown about to be struck by hammer 6 during its downward movement after being release from sear 17 (see Fig. 1b and 3a, 3b). Also shown is right side portion 10a and left side portion 10b of device 10. Fig. 1c further shows decocking portion surface 34b of portion 10b and surface 27b of portion 10a.

Turning to Fig. 2, there is shown right side portion 10a, left side portion 10b, firing pin 16 with front end 16a, central head 16h, vertical head surface 16v, curved head and cam surface 16w, rear portion 16c and right side projection 16p. Sear 17, pivotal about axis 17a, has upper arm 171 which is acted on by decocker portion 10b as later described. Upper block piece 22 engages vertical surface 16v of firing pin 16 to prevent forward movement of the pin 6 against the cartridge (not shown). Lower blocker unit 18 pivotal about axis 18a has lever arm 181 for raising upper firing pin block piece 22 to permit firing pin vertical surface 16v to move forward under raised block piece 22.

Upper block crosspiece 22 which includes vertical stem piece 22v and plunger portion 22c is urged downwardly by a plunger assembly 22a comprised of spring 22b and plunger 22c which is contained by rear sight 9. Blocker piece 22 is sufficiently forward of vertical surface 16v even when in its lower position to permit pin 16 to be moved forward by device 10 as further described. Lower blocker unit 18 carries a side foot piece 18f for positioning in opening 19 of trigger bar 20. When trigger 20 moves rearward during the last portion of trigger pull, blocker unit 18 is rotated causing lever arm 181 to lift firing pin block crosspiece 22 to a point where crosspiece 22 is clear of vertical surface 16v of firing pin 16 just prior to firing. Right side portion 10a includes body 25, wing thumb piece 25a, portion engageable prong 25b and lock groove 25c. Elongated lock piece 26 is also shown which, as assembled, is positioned in and secured to slide 5 with end 26a in slot 25c to hold portion 10a in slide side hole 5a while permitting partial rotation of portion 10a therein. Also shown is a hammer 6 and trigger 21.

Further with respect to Fig. 2, left side portion

10b includes left body 29, thumb piece or wing 29a, protrusion-receiving slot 29b, spring-receiving recess 29c with cross rod 31 mounted therein (see also Fig. 5). As assembled, portion 10a and portion 10b rotate as a unit providing ease of assembly and ambidextrous operation.

Turning now to Fig. 3a, left portion 10b with its body 29, and its wing 29a slot 29b has rear surface 34 comprised of surface portions 34a, 34b. Also shown is lower sear cam surface 32 comprising surface portion 32a, 32b. Sear arm 18b nests against surface 32. Third cam surface 35 comprising surface portions 35c, 35b, nests against firing pin surface 16w. Hammer engaging firing pin portion 16c, when in its ready-to-fire position extends into slide cutout 14 beyond and to the rear of vertical hammer seating surfaces 15. Firing pin 16 is urged to the rear against a stop by firing pin spring (neither of which are shown) so that hammer engaging portion 16c is properly exposed to hammer 6 during its downward and forward movement as pistol 10 is fired.

In Fig. 1b, hammer 6 is released as lower cam surface 12 rotates out of its nested position thus moving sear arm 17b downwardly. Also, as portion 10b rotates its cam surface 15a and surface 29s of body 29 act on firing pin surface 16w to move firing pin 16 forward. Firing pin rear portion 16c is thereby moved to the left of lower surface 15b thus protecting pin 16 from being struck and propelled forward by hammer 6.

Turning to Fig. 4, body 25 of right portion 10a includes internal curved pocket 28 and opening 30 in the side pocket 28. Pocket 28 engages firing pin right projection 16p as portion 10a is rotated counterclockwise to prevent projection 16p (and pin 16) from moving further in the forward direction. Opening 31 functions to permit pin side projection 16p to move forward during firing. Note portion 16c is x distance to the rear of lower surface 15b before decocker rotation and is y distance forward of lower surface 15b after rotation of the decocker 10 and the camming forward by portion 10b as described above.

Finally, in Fig. 5, spring retaining recess 41 in slide 5 accommodates return spring 38 which rests under pin 31 in slot 29c of body 29 of portion 10b. Pin 39 holds spring 38 in position. The phantom lines show decocker portion 10b rotated to depress spring 38.

In operation, device 10 is rotated from its at rest position through its arcuate range of about 45° including intermediate activating positions until it reaches its fully activated position (Figs. 3b and 4) and is then released for return by spring 38 (Fig. 5) or, alternatively, held in its active position. During movement with spring return the following occurs:

(1) firing pin 16 through its surface 16w is cammed forward until the rearward end 16c of pin 16 is moved forward of slide surface 15b;

(2) when (1) above has been accomplished, sear 17 is thereafter rotated by cam surface 32 acting on sear arm 17b to rotate sear 17 and to release hammer 6 which is urged downwardly against surface 15b permitting decocking without discharge of pistol 10; and

(3) throughout rotation of device 10 surfaces 34a, 34b and 27a, 27b remain forward of seating surface 15b;

(4) as hammer 6 is released, cam surface 28 prevents firing pin 16's forward movement by engaging and blocking further forward movement of pin projection 16p;

(5) the device 10, as released by the operator, returns to its rest position leaving hammer 6 against surface 15b, surfaces 29 and 35 rotate allowing the firing pin 16 to move rearward and slot 31 is oriented to permit firing pin 16 to thereafter move forward against the cartridge when the hammer strikes pin 16 during subsequent normal firing sequences.

During operation without spring return, device 10 is held in its active firing-pin-forward position until the operator rotates device 10 back to permit the firearm to be operated in the firing mode.

Claims

1. In a firearm having a slide with a rearward hammer seating surface which limits downward movement of the hammer when the firearm is fired, a firing pin having a forward end for engaging the cartridge and a rearward end normally extending rearward of the seating surface and a sear engageable with a hammer to hold the hammer in a cocked position, the improvement comprising

1) a transverse rotatable device movable through an arc from an inactive position to active positions;

2) sear engagement cam means on the rotatable device for engaging the sear to rotate the sear to release the hammer at a selected active position; and

3) firing pin engagement cam means on the rotatable device for moving the pin forward so that it is positioned forward of the slide seating surface a selected distance

whereby upon rotation of said device the surface means as positioned rearward of the firing pin prevents the hammer from striking the firing pin upon its release from the sear.

2. The firearm transverse rotatable device of claim 1 in which the device includes a rotatable body in turn comprising two interconnecting body portions mounted transversely in the slide and in

which sear engagement cam means and pin engagement means are part of one such body portion.

3. The firearm transverse rotatable device of claim 2 having in addition a surface on the other body portion that limits forward movement of the firing pin during decocking. 5

4. The firearm transverse rotatable device of claim 1 in said device remains in an active position to hold the firing pin forward. 10

5. In an automatic firearm having a hammer, a rotatable sear, a slide and a rear hammer down seating surface on the slide, a transverse rotatable device having a rest position and a number of activating positions comprising 15
 - a) first and second body portions shaped to interconnect with one another, with each portion being substantially cylindrical in shape, such portions in turn having 20
 - i) surface means on each portion positioned spaced from and forward of said seating surface when the device is in its various positions; 25
 - ii) a lower sear camming surface on the first portion to cam the rotatable sear;
 - iii) a firing pin camming surface in the first portion to cam the firing pin forward; and 30
 - iv) a surface on the second portion to prevent movement of the firing pin forward of a selected point during decocking.

6. The device of claim 5 in which the first body portion has a hand operable lever means extending from one of its sides. 35

7. The device of claim 5 in which the first portion has an internal spring means receiving recess and spring means positioned in the spring-receiving recess for causing the lever to return to its inactive position. 40

8. The device of claim 5 in which the holding means holds the lever in a position which causes the firing pin to be positioned forward. 45

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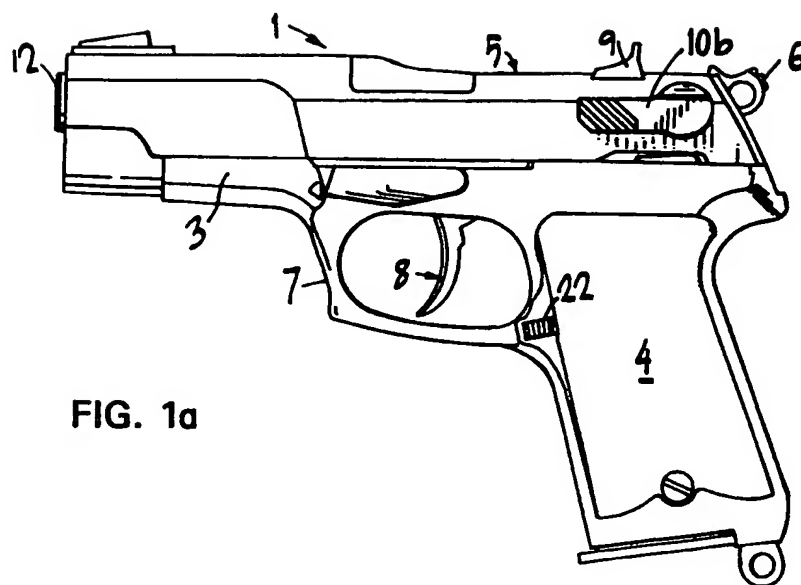


FIG. 1a

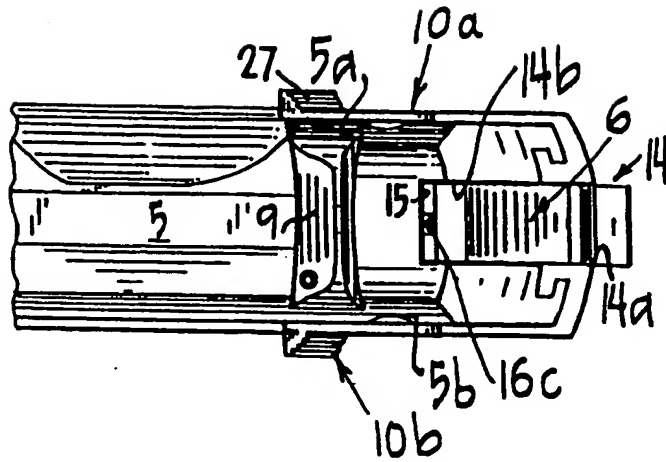


FIG. 1b

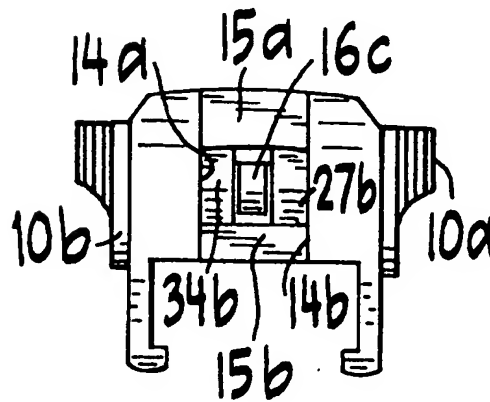


FIG. 1c

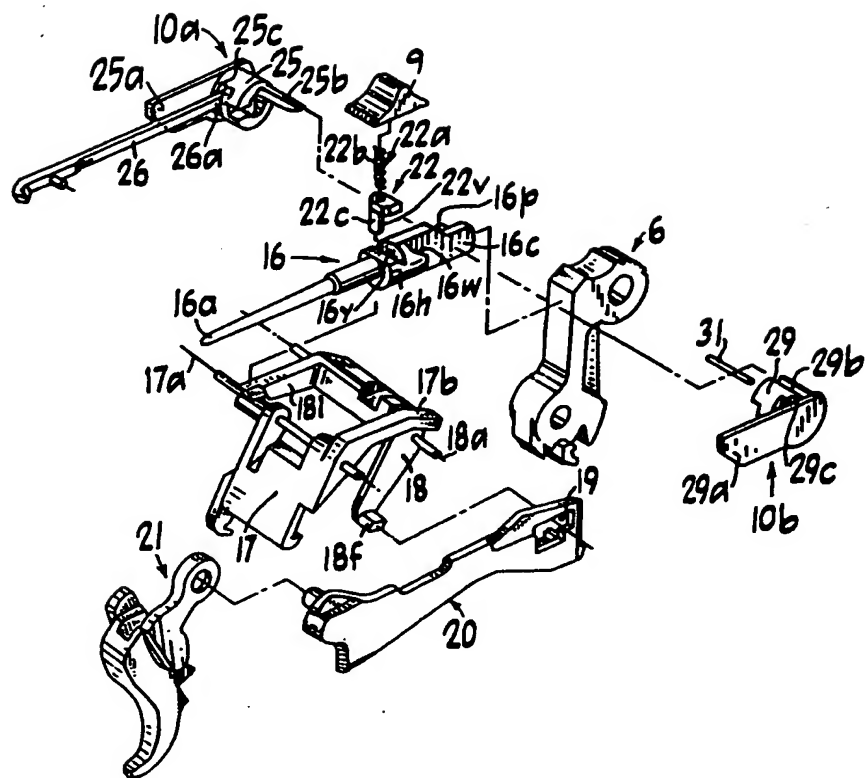


FIG. 2

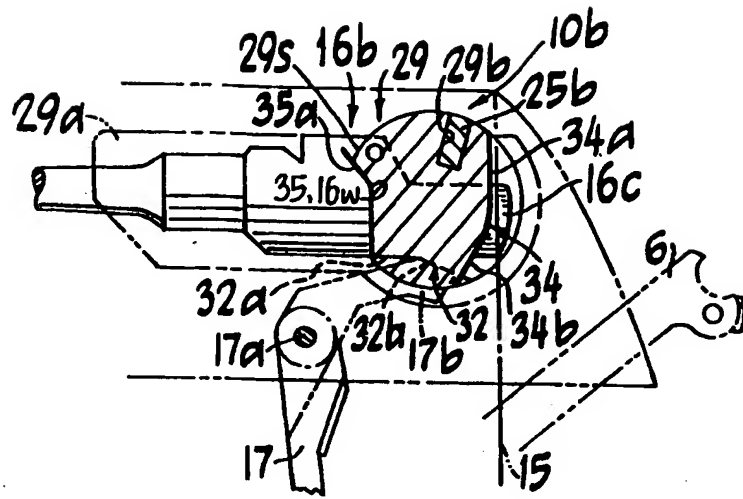


FIG. 3a

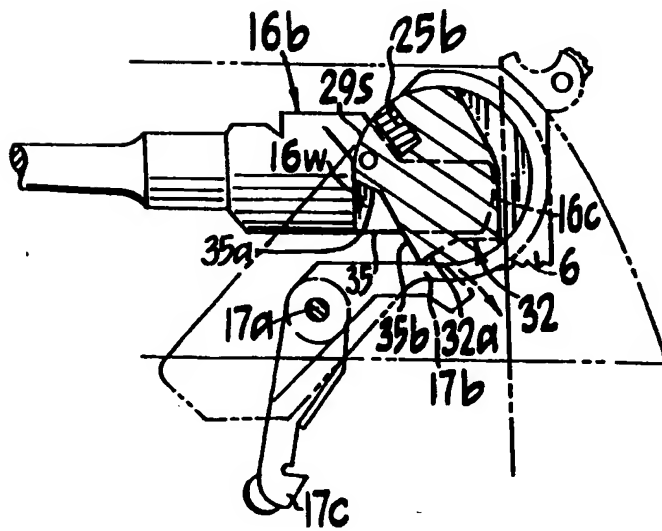


FIG. 3b

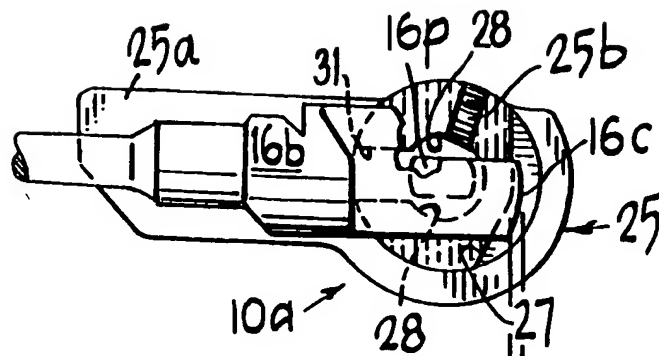
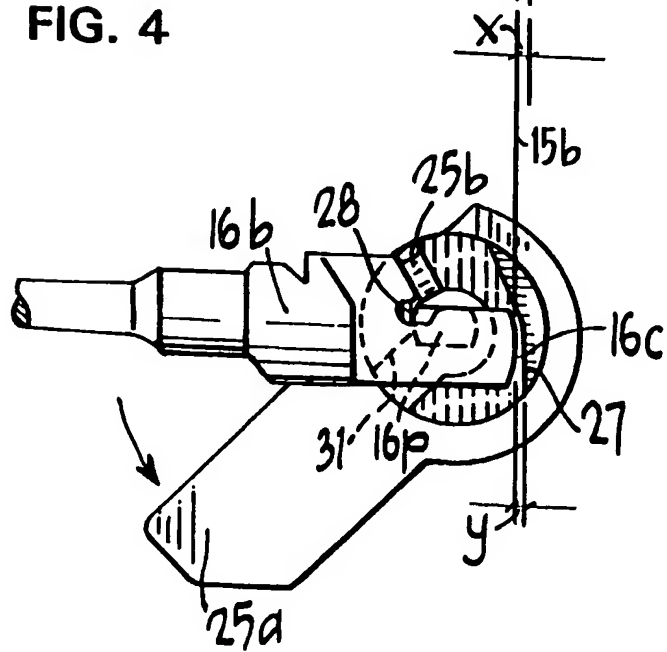


FIG. 4



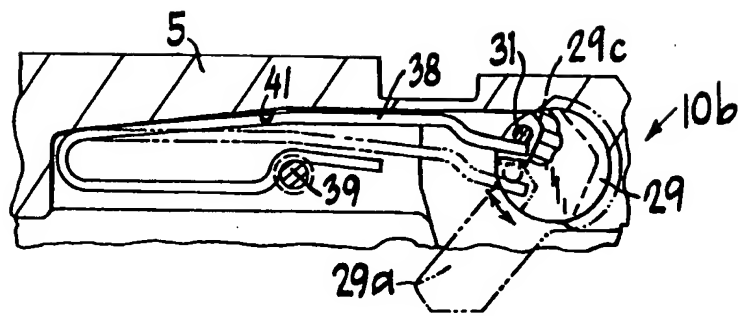


FIG. 5



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EUROPEAN SEARCH REPORT

Application Number

EP 92 30 7358

| DOCUMENTS CONSIDERED TO BE RELEVANT | | | |
|--|---|--|--|
| Category | Citation of document with indication, where appropriate, of relevant passages | Relevant to claim | CLASSIFICATION OF THE APPLICATION (Int. Cl. 5) |
| A | FR-A- 634 617 (WALTHER) * Page 2, line 29 - page 3, line 2; figures 1-5 * | 1,5 | F 41 A 17/74 F 41 A 19/52 |
| A | US-A-4 536 981 (GIRAGOSIAN) * Column 4, lines 46-60; figures 2-8,12 * | 1,5 | |
| A,D | US-A-4 589 327 (SMITH) * Abstract; figures * | 1,5 | |
| A,D | US-A-4 590 697 (RUGER) * Abstract; figures * | 1,5 | |
| A | GB-A- 460 859 (MAUSER) * Page 2, lines 43-129; figures 1,2 * | 1,5 | |
| A | US-A-2 373 213 (WILLIAMS) | | |
| | | | TECHNICAL FIELDS SEARCHED (Int. Cl. 5) |
| | | | F 41 A |
| The present search report has been drawn up for all claims | | | |
| Place of search THE HAGUE | | Date of completion of the search 26-11-1992 | Examiner RODOLAUSSE P.E.C.C. |
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